

Prestressed Steel Structures*

By G. Magnel, M.I.Struct.E.

*Professor at the University of Ghent.
Member of the Royal Academy of Belgium.*

Introduction

The title of the present paper will be surprising to many engineers who, being familiar with prestressed concrete, will not understand at first sight what can justify the application of the same principles to constructions made in structural steel.

The main reason generally given for prestressing concrete is the poor tensile resistance of this material; however, prestressing of concrete gives other advantages than the one consisting of cancelling the tensile stresses produced by the loads; it allows the use of higher working stresses in the concrete; it does away with the difficulties in relation to resistance to shear; it permits lighter constructions, needing only half of the concrete and one-sixth of the steel as compared to reinforced concrete; that steel costs only three times more per unit weight than ordinary reinforcing bars.

Some of these advantages can be obtained by prestressing steel. Indeed, the working stress of mild steel, say 20,000 p.s.i., may be compared with the 140,000 p.s.i. allowable for the high tensile steel wires we are proposing to use. Hence, to resist a given force with high tensile steel, only one-seventh of the weight necessary with mild steel will be required; as on the other hand high tensile steel costs only three times as much per unit weight as mild steel, the conclusion is that replacing the former by the latter produces an economy of 86 per cent. in weight and of 57 per cent. in cost.

There is of course no question of suggesting such a substitution in practice for tensile members, as the deformation would be multiplied by seven and as it would become impossible to fix any other element to these tensile members made of wires.

Our only purpose is to illustrate by a simple statement what saving can be effected, if one goes to the unpractical limit.

Our practical proposal however is more modest, although it will allow us to save as much as 50 per cent. in weight and 20 per cent. in cost, without decreasing the factor of safety nor increasing unduly the deformations.

The general idea will be to replace a tensile member—say of a bowstring girder—requiring, if made in mild steel, a cross-section A , by another one made of mild steel with a reduced cross-section A_r but prestressed by means of a cable with a cross-section A_c .

We are going to show that it is possible to choose A_r and A_c in such a way that the safety factor remains the same as in the classical mild steel construction and that the deformation of the member remains inside acceptable limits.

We intend using the following notations for the permissible steel stresses:

t for tensile stresses in members made in ordinary mild steel;

t' for compressive stresses in a short member in ordinary mild steel; short enough for the danger of buckling to be forgotten;

t_1 for tensile stresses in the mild steel of a prestressed member composed of a section A_r in mild steel prestressed by means of a cable having A_c as cross-section;

t_c for the tensile stresses in the cable producing the prestressing.

At first sight the reader will not see why we use two different notations t and t_1 for the permissible tensile stress of mild steel. This will be explained clearly later.

Preliminary Problem

Let us consider a tensile member having to resist a force that varies from zero to F .

Let P_1 be the initial prestressing force given by the cable at the moment where the cross-section A_r is put under prestress; let t_{wc} be the working stress in the cable at that moment (w for working).

When the prestressing is finished and the force F is applied to the prestressed member, this force is resisted partly by the mild steel A_r and partly by the cable A_c ; let ΔP_1 be the increase of P_1 due to F . In order to find the parts of F taken up by the mild steel and by the cable, it is sufficient to write that their strains are the same.

Hence we have the following formulæ:

$$P_1 = A_r t' = t_{wc} A_c \quad (1)$$

$$\Delta P_1 = \frac{\alpha}{1 + \alpha} F \text{ if } \alpha = \frac{A_c}{A_r} \times \frac{E_c}{E} \quad (2)$$

$$P_1 + \Delta P_1 = A_c t_c \quad (3)$$

$$F - P_1 - \Delta P_1 = A_r t_1 \quad (4)$$

By solving these four equations we find, if $\beta = E : E_c$

$$A_c = \frac{F}{t_1 + t'} \times \frac{\beta t'}{\beta t_c - t_1} \quad (5)$$

$$A_r = \frac{F}{t_1 + t'} \times \frac{\beta t_c - t_1 - t'}{\beta t_c - t_1} \quad (6)$$

$$P_1 = \frac{F}{t_1 + t'} \times \frac{\beta t_c - t_1 - t'}{\beta t_c - t_1} \times t' \quad (7)$$

$$\Delta P_1 = F \frac{t'}{\beta t_c - t_1} \quad (8)$$

It is now easy to calculate:

a.—the relative weight $W_1 : W$ of the unit length of the prestressed member (W_1) to the one of the classical member in mild steel (W).

b.—the relative cost $C_1 : C$.

c.—the relative elongation $\Delta_1 : \Delta$.

*Paper to be read before the Institution of Structural Engineers at 11, Upper Belgrave Street, London, S.W.1, on Thursday, November 9th, 1950, at 6 p.m.

d.—the factor of safety S based on the yield point of the mild steel, supposed to be equal to 2 t.

It is found

$$\frac{W_1}{W} = \frac{t}{t_1 + t'} \times \frac{\beta t_c - t_1 - t' + \beta t'}{\beta t_c - t_1} \quad (9)$$

$$\frac{C_1}{C} = \frac{t}{t_1 + t'} \times \frac{I}{\beta t_c - t_1} \times \left(\frac{c_c}{c} \right) \quad (10)$$

(c_c and c are the costs of the unit weight of the wires and the mild steel).

$$\frac{\Delta_1}{\Delta} = \frac{t_1 + t'}{t} \quad (11)$$

$$S = \frac{2t + t'}{t_1 + t'} \quad (12)$$

We will now re-write these formulæ in the supposition that

$$t' = t = 20,000 \text{ p.s.i.}$$

$$t_c = 140,000 \text{ p.s.i.}$$

$$c_c : c = 3 \quad \beta = 1$$

It is found

$$\frac{W_1}{W} + \frac{20,000}{t_1 = 20,000} \quad (13)$$

$$\frac{C_1}{C} = \frac{20,000}{t_1 + 20,000} \left(1 + \frac{40,000}{140,000 - t_1} \right) \quad (14)$$

$$\frac{\Delta_1}{\Delta} = \frac{t_1 + 20,000}{20,000} \quad (15)$$

$$S = \frac{60,000}{t_1 + 20,000} \quad (16)$$

We can extract from these formulæ the table I.

TABLE I

	$t_1 = 10,000 \text{ p.s.i.}$	$t_1 = 20,000 \text{ p.s.i.}$
$\frac{W_1}{W}$	0.67	0.50
$\frac{C_1}{C}$	0.87	0.66
S	2.00	1.50
$\frac{\Delta_1}{\Delta}$	1.50	2.00

On diagrams I, A, B, C and D we have drawn the curves giving the four interesting values in function of t_1 .

It is now clear that 33 per cent. in weight and 13 per cent. in cost can be gained without decreasing the normal safety factor of two, and by adding only 50 per cent. to the elongation; this result is obtained by adopting t_1 as low as 10,000 p.s.i.

As a numerical example let us mention the following figures for $F = 100 \text{ t.} = 224,000 \text{ lb.}$

$$A_c = 1.15 \text{ sq. in.} \quad A_r = 6.32 \text{ sq. in.}$$

$$P_1 = 125,400 \text{ lb.} \quad \Delta P_1 = 33,300 \text{ lb.}$$

(26.5 per cent. of P_1)

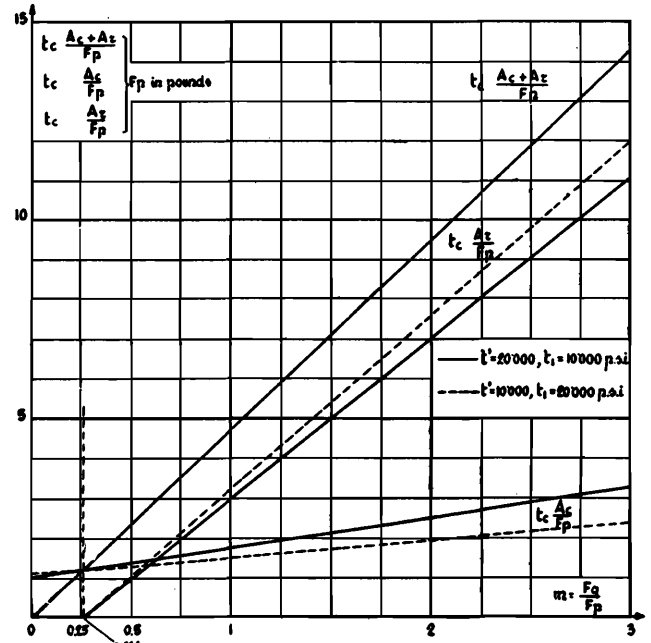


Diagram I

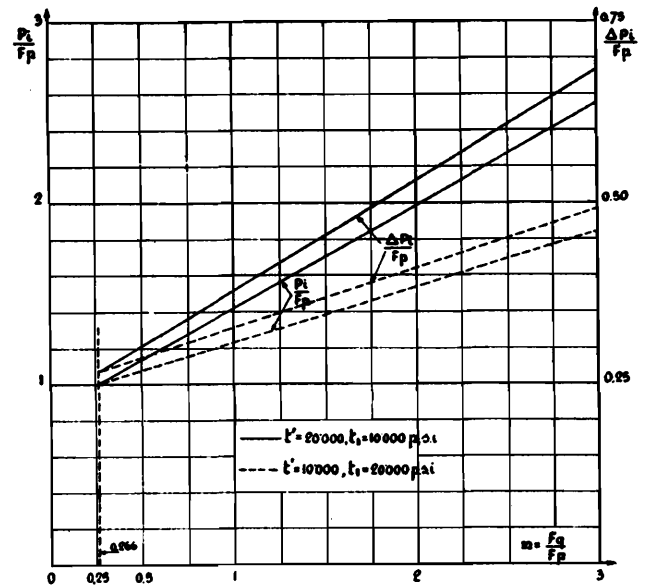


Diagram II

Let us mention that the classical member in mild steel requires $A = 11.2 \text{ sq. in.}$

We can go much further in the direction of economy. Let us choose a cross-section of mild steel still smaller than A_r and prestress it so that it is capable of carrying only a part of F equal to F/n ; then when the member carries this force let us prestress it further until the compressive stress becomes again t' ; let us then apply

another $F : n$ so as to create a tensile stress t ; let us then prestress it again and so on.

The relative weight becomes

$$\frac{W_n}{W} = \frac{t}{n(t_1 + t')}$$

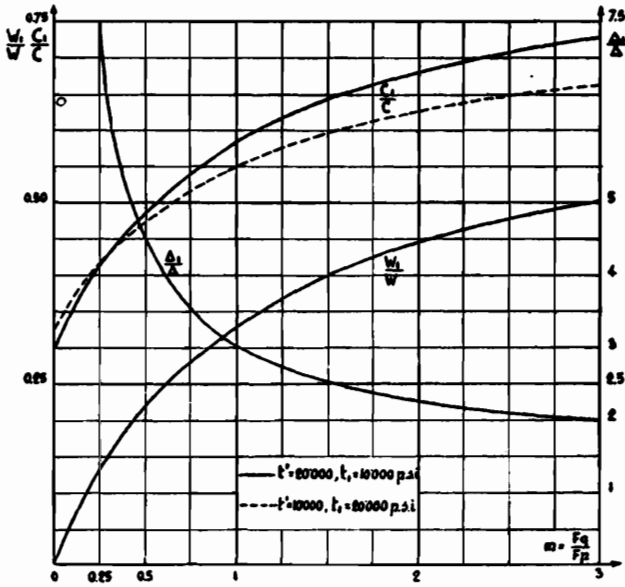


Diagram III

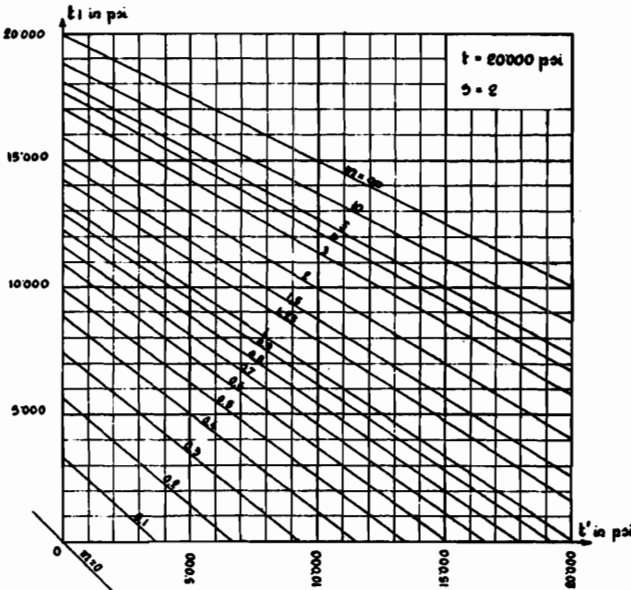


Diagram IV

We have not mentioned this method in view of its application, as the above table shows that the factor of safety decreases as n increases from $n = 1$ onward.

The Effect of the Decrease in Weight

In the above elementary theory we have assumed that the decrease in weight of the tensile member, from W to W_1 had no effect on the load F to be resisted. But this is not so in practice.

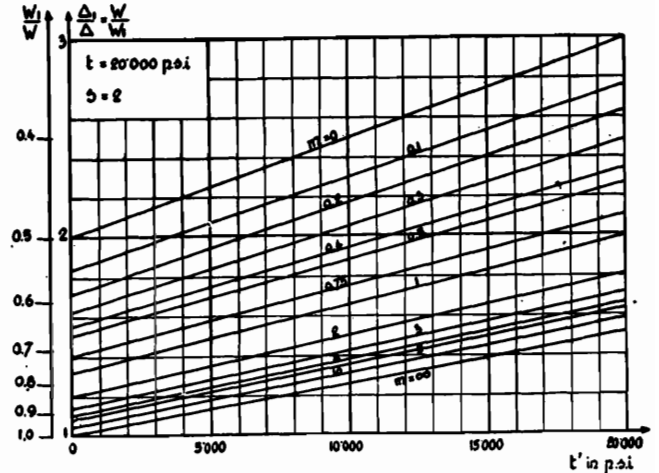


Diagram V

Let us take as example the case where the tensile member is the lower member of a bowstring girder; let us also assume that in the classical design the weight of the tensile member is one-third of the weight of the girder; gaining 33 per cent. in the weight of the tensile member means gaining 11 per cent. in the total weight of the girder.

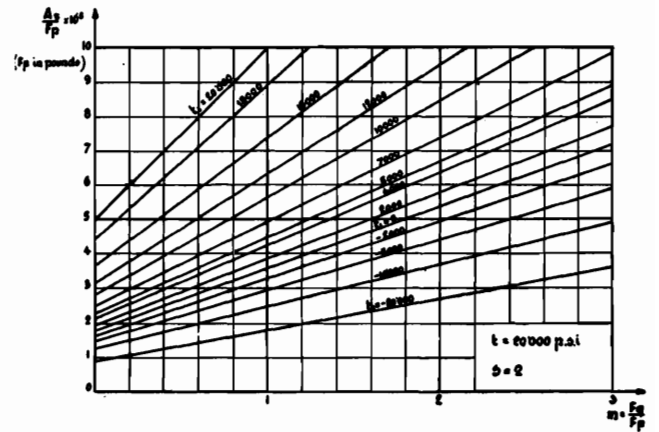


Diagram VI

The following table gives numerical results obtained by this way of working:

	A_c	A_r	S	$\frac{W_n}{W}$	$\frac{C_n}{C}$	$\frac{\Delta_n}{\Delta}$
$n = 0$	0	11.2	2.00	1.00	1.00	1.00
1	1.15	6.27	2.00	0.67	0.87	1.50
2	1.39	2.30	1.50	0.33	0.58	1.50
3	1.48	0.98	1.33	0.22	0.49	1.50
4	1.53	0.314	1.25	0.17	0.44	1.50
4.76	1.55	0	—	0.14	0.42	—

(This table is made in the supposition that $t = t' = 20,000$ p.s.i.; $t_1 = 10,000$ p.s.i.; $t_c = 140,000$ p.s.i.)

This assumes that a fraction $F : n$ of F is the only changeable part in the total load.

Let us assume now that in general p per cent. is gained in the weight of the bowstring girder by prestressing its lower member; we have then

$$W_p = W_m \left(1 - \frac{p}{100} \right)$$

if W_m is the weight of the girder in mild steel and W_p the weight of the girder with its prestressed member.

Let Q be the working load acting on the girder beside its own weight.

The percentage p_t gained in the total load to be carried is

$$p_t = p \frac{W_m}{W_m + Q}$$

This saving in total weight reduces in the same proportion the force F to be carried by the prestressed member, and it reduces also the forces to be carried by all the elements of the girder; consequently all the cross-sections of the elements of the girder can be reduced so that its weight becomes

$$W'_p = W_p \left(1 - \frac{pt}{100} \right)$$

This reduction allows a further reduction in cross-sections, so that the weight becomes

$$W''_p = W'_p \frac{W'_p + Q}{W_p + Q}$$

and again

$$W'''_p = W''_p \frac{W''_p + Q}{W'_p + Q}$$

and so on.

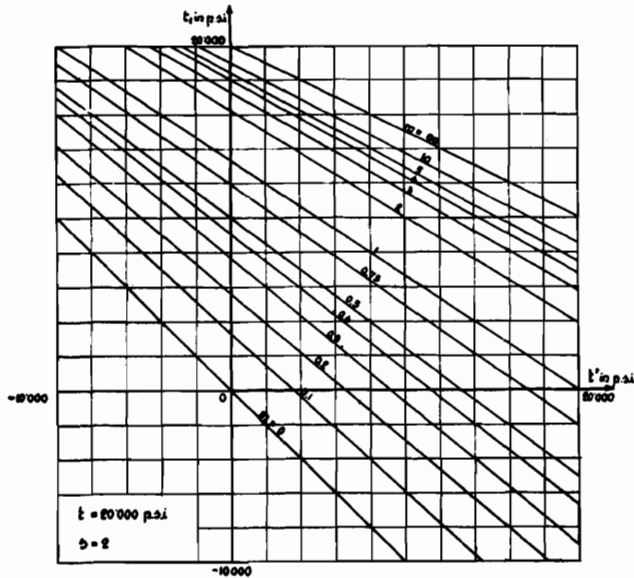


Diagram VII

If we compute the limit of these figures, it is found

$$W_p^\infty = W_m \frac{1 - \frac{pt}{100}}{\frac{Q}{W_m} \times \frac{1}{1 - \frac{p}{100}} + \frac{pt}{100}} \times \frac{Q}{W_m}$$

The following table gives numerical values in the case where

$$t = t' = 20,000 \text{ p.s.i.} \quad t_1 = 10,000 \text{ p.s.i.}$$

$$t_c = 140,000 \text{ p.s.i.}$$

$$Q : W = 0.25 \quad Q : W = 1$$

% economy in weight of lower member	33.3	33.3
% economy in weight of beam	11.1	11.1
% final economy in weight	38.4	20.0
% economy in cost of lower member	13.0	13.0
% economy in cost of beam	4.3	4.3
% final economy in cost	18.3	8.2

The writer is aware of the fact that the above calculation is not quite correct, as the weight of a girder cannot be exactly reduced in the same proportion as the total load to be carried. But this slight error is compensated by the fact that we neglected other sources of economy; in a trussed girder, for example, the cable can be raised towards the supports and a shearing force of the contrary sign of those given by the loads introduced; this allows the cross-section of diagonals and vertical elements to be reduced; in a girder with a solid web the

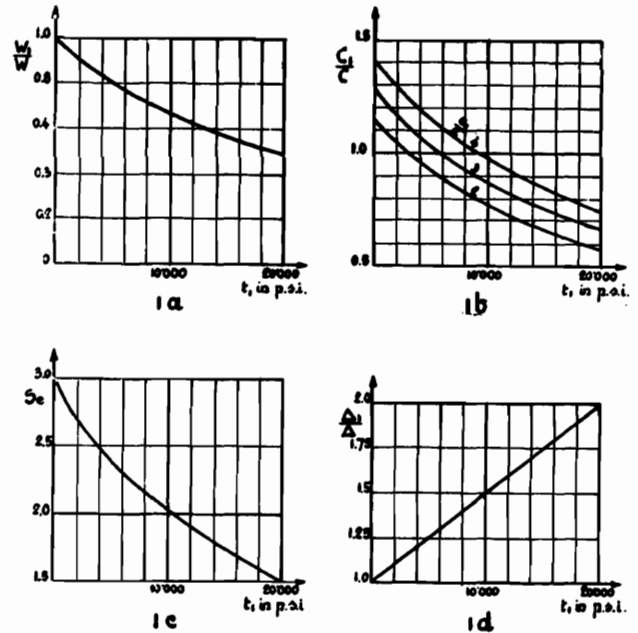
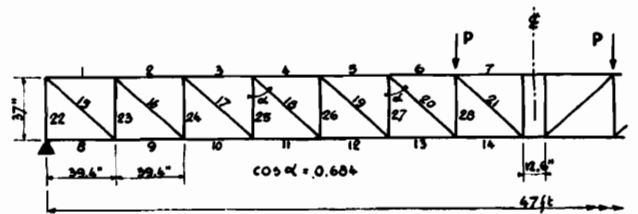


Fig. 1.—Diagrams corresponding to formulae 13 to 16.

prestressing of the lower member produces tensile stresses in the top member, which allows a reduction in its cross-section.

The conclusion is that a considerable saving in weight and cost can be made by prestressing the tensile member of a bowstring girder; this saving is the most important when the dead weight of the girder is considerable as compared to the additional load acting on it. This mainly happens for very big spans and light live loads, as for example in the beams covering an aeroplane hangar of 500 ft. span.



Theoretical shape of the girder

Fig. 2

Fundamental Differences between Prestressed Steel and Prestressed Concrete

In a prestressed concrete beam, the stress in the cable varies only by 3 to 4 per cent. when the additional load acts on the beam. This is not true for a prestressed steel girder; in prestressed steel these variations are three to four times higher; this is due to the fact that

the stresses involved in the mild steel A_r are about twenty times higher than for concrete, whereas the modulus of the mild steel is only five to six times that of concrete.

A second difference is that in prestressed concrete, stretching two wires at a time does not mean a loss in average prestress of more than about 4 per cent., this loss being due to the fact that every new pair of wires which is stretched, makes a loss in prestress of those stretched previously. In prestressed steel this effect is much larger and comes to about nine or ten per cent., which must be taken into account in practice.

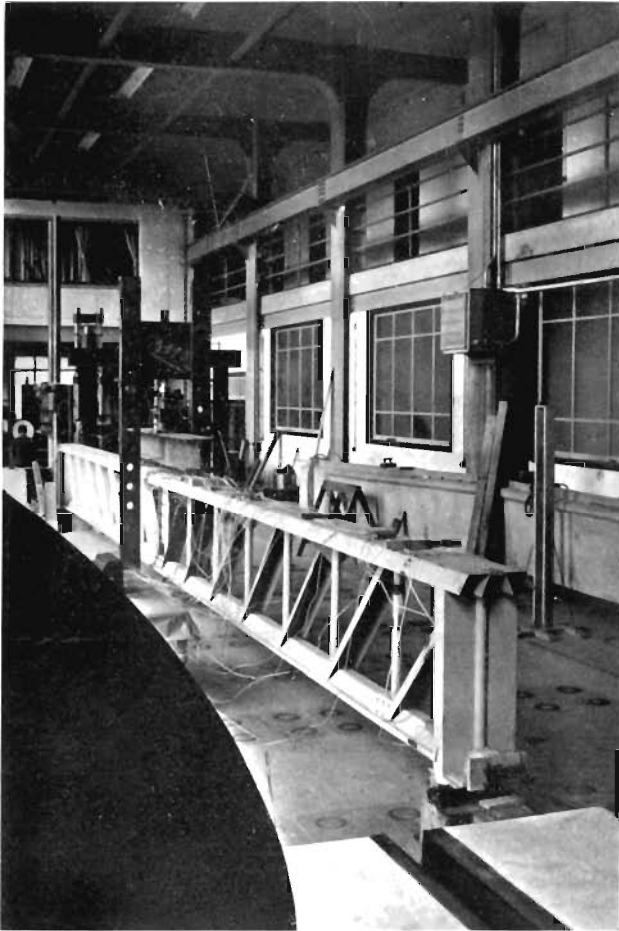


Fig. 3.—General view of the girder

Finally, let us point to the fact that there is no question here of loss of prestress through shrinkage, nor even of creep of the mild steel, as the stresses to which it is submitted are lower than half the yield stress; finally, with the modern wire, there is practically no relaxation.

Design of a Tensile Member having to Resist both Permanent and Non-Permanent Forces

This is the real practical problem: a member carries permanently F_p and beside this must be capable of carrying a load varying from zero to F_q .

We admit that F_p comes in gradually as the prestressing force P_1 increases from zero to P_1 . The method of construction must be such that this condition is practically observed.

It is very easy to establish the formulæ relative to this problem; they are

$$P_1 = F_p + t' A_r \tag{17}$$

$$F_q - \Delta P_1 = (t' + t_1) A_r \tag{18}$$

$$P_1 + \Delta P_1 = t_c A_c \tag{19}$$

By solving them, we find

$$A_r = \frac{F_p}{t_c - t_1} \left[m \frac{t_c - (t' + t_1)}{t' + t_1} - 1 \right] \tag{20}$$

$$A_c = \frac{F_q}{t' + t_1} - A_r \tag{21}$$

$$P_1 = F_p + t' A_r \tag{22}$$

On the other hand

$$\Delta P_1 = F_q \frac{\alpha}{1 + \alpha} \tag{23}$$

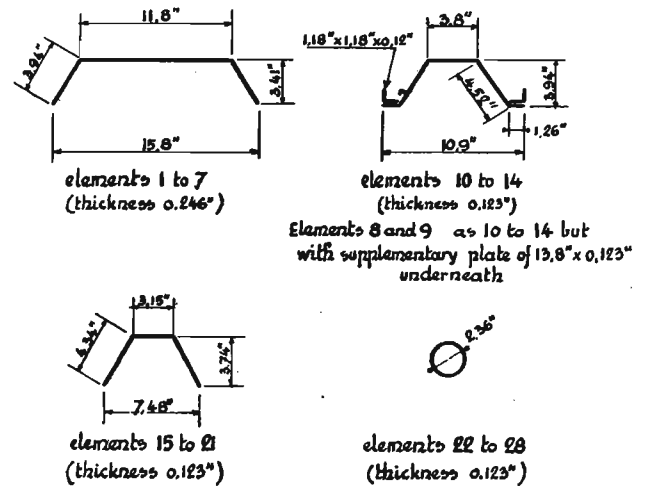


Fig. 4

(we put $\alpha = \frac{A_c}{A_r}$ and $m = \frac{F_q}{F_p}$).

To be noted that equation (21) can be transformed in

$$A_r + A_c = \frac{F_q}{t' + t_1}$$

which shows that the total steel area—the total weight—is independent of the permanent load and depends only on the live load. Something similar is known in prestressed concrete, where it is generally said that “the dead load carries itself”; however, such a statement is only clear after a long explanation which can be found in our book on prestressed concrete.

The last formula does not mean that, when $F_q = 0$ we do not require any cross-section $A_c + A_r$ to carry F_p . Indeed, in the cases where the formulæ (20) to (22) give either negative values for A_r or small positive ones, we should not accept these results; let us not forget that a tensile member, in a bowstring girder for example, has another function besides resisting tensile forces; it must have a certain side stiffness and allow the fixing to it of

the elements making the bridge deck or the platform carried by this girder.

It is easy to compute the formulæ for relative weight, cost and elongation :

$$\frac{W_1}{W} = \frac{t}{t' + t_1} \times \frac{m}{1 + m} \quad (24)$$

$$\frac{C_1}{C} = \frac{W_1}{W} + \frac{2t}{t_c - t_1} \left(1 - \frac{t_1}{t} \times \frac{W_1}{W} \right) \quad (25)$$

or

$$\frac{C_1}{C} = \frac{t}{(1 + m)(t' + t_1) \left(m + \frac{2(m + 1)t' + 2t_1}{t_c - t_1} \right)} \quad (26)$$

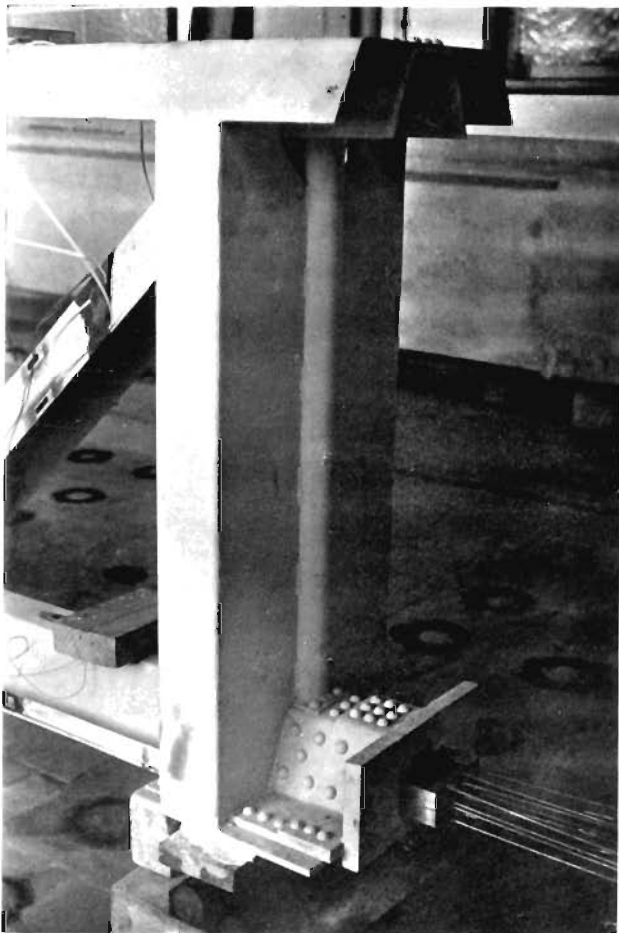


Fig. 5.—Anchorage of the wires at the end of the girder

finally

$$\frac{\Delta_1}{\Delta} = \frac{t_1 + t_1}{t} \times \frac{1 + m}{m} \quad (27)$$

Noted that $\frac{\Delta_1}{\Delta}$ is the reverse from $\frac{W_1}{W}$; to be re-

membered that the Δ are the elongations due to F_q and not due to $F_p + F_q$; this is because the only thing that matters generally in practice is the deformation due to F_q .

We have made diagrams representing formulæ (20) to (27) ; on these diagrams I, II and III we have adopted $t_c = 140,000$ p.s.i. and $t = 20,000$ p.s.i. ; two cases of values for t' and t_1 have been considered ; they are such that $t_1 + t' = 30,000$ p.s.i. in both cases.

The following conclusions can be drawn from these diagrams :

- It is more economical to adopt $t_1 = 10,000$ p.s.i. than 20,000 p.s.i.
- In both cases considered the values of $W_1 : W$ and $\Delta_1 : \Delta$ are the same.
- A_r is zero for $m = 0.266$. If m is smaller or near to this value the solution given by the diagrams is not acceptable, as already pointed out.
- For $m = 1$ or more the value of $\Delta_1 : \Delta$ is acceptable and the economy in weight and cost is considerable ; for example with $m = 1$ we gain 67 per cent. in weight and 45 per cent. in cost.

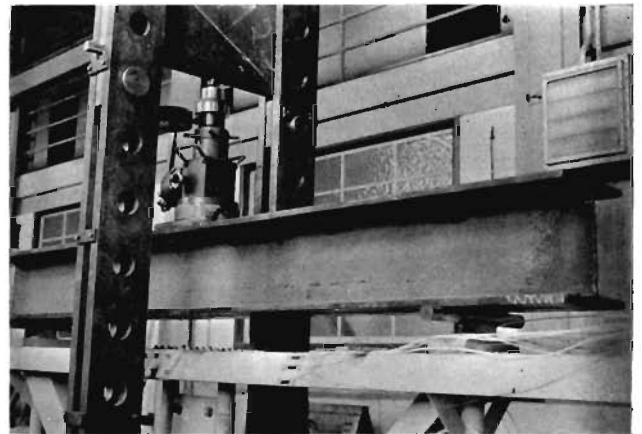


Fig. 6.—The loading system

Coming back to the conclusion "c" we will explain how to solve the problem in cases where the diagram gives a value of A_r which is not acceptable. In that case a minimum value must be chosen for A_r , taking into account the practical conditions of the problem. It is obvious that having chosen a A_r larger than the one given by the diagrams, there will be no need to put it under compression up to the permissible stress t' ; let us call t'_w the stress to be used (working stress).

The formulæ are then

$$A_c = \frac{F_p + F_q - t_1 A_r}{t_c} \quad (28)$$

$$P_1 = t_c A_c - F_q \frac{A_c}{A_c + A_r} \quad (29)$$

$$t'_w = \frac{P_1 - F_p}{A_r} \quad (30)$$

$$\Delta P_1 = F_q \frac{A_c}{A_c + A_r} \quad (31)$$

$$\frac{\Delta_1}{\Delta} = \frac{t'_w + t_1}{t} \times \frac{1 + m}{m} \quad (32)$$

Should it be found that the elongation is exaggerated, the problem would have to be re-examined by taking the acceptable value of $\frac{\Delta_1}{\Delta}$ as data.

The formulas are then

$$t'_w = \frac{\Delta_1}{\Delta} \times \frac{m}{1+m} t - t_1 \quad (33)$$

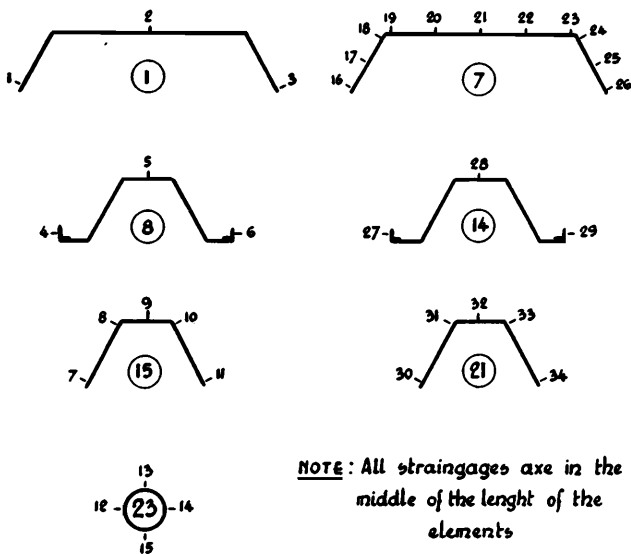
$$A_r = \frac{F_p}{t_c - t_1} \left[m \frac{t_c - (t'_w + t_1)}{t'_w + t_1} - 1 \right] \quad (34)$$

$$A_c = \frac{F_q}{t'_w + t_1} - A_r \quad (35)$$

$$P_1 = F_p + t'_w A_r \quad (36)$$

$$\Delta P_1 = F_q \frac{A_c}{A_c + A_r} \quad (37)$$

$$\frac{W_1}{W} = \frac{t}{t'_w + t_1} \times \frac{m}{1+m} = \frac{\Delta}{\Delta_1} \quad (38)$$



NOTE: All straingages are in the middle of the length of the elements

Fig. 7

$$\frac{C_1}{C} = \frac{t}{(1+m) \frac{(t'_w + t_1)}{2(m+1)t'_w + 2t_1}} \quad (39)$$

or

$$\frac{C_1}{C} = \frac{W_1}{W} + \frac{2t}{t_c - t_1} \left(1 - \frac{t_1}{t} \frac{W_1}{W} \right) \quad (40)$$

The above theory is still incomplete, as it does not take into account the factor of safety.

Safety Factor

Let us admit that the mild steel has a yield point of $2t$; in other words, the safety factor for the member made entirely in mild steel is 2.

Let us compute the safety factor of the prestressed member calculated with formulas (20) to (22).

The effect of F_q is to change a compressive stress t' in a tensile stress t_1 ; the reserve of stress existing then before the yield point is reached is $2t - t_1$; this means that to $F_p + F_q$ a force F must be added equal to

$$F = F_q \frac{2t - t_1}{t' + t_1} \quad (41)$$

before the yield point is reached. Hence

$$S = \frac{F_p + F_q + F}{F_p + F_q} \quad (42)$$

or

$$S = 1 + \frac{m}{1+m} \times \frac{2t - t_1}{t' + t_1} \quad (43)$$

In this case we find

$$\frac{W_1}{W} = \frac{t}{t' + t_1} \times \frac{m}{1+m} = \frac{t(S-1)}{2t - t_1} \quad (44)$$

If one accepts $S = 2$ equation (43) becomes

$$t_1 = \frac{m(2t - t') - t'}{1 + 2m} \quad (45)$$

In other words, if it is desired that the safety factor for the prestressed member be 2, there is a relation between t' and t_1 . Our diagram IV translates formula (45) in the case where $t = 20,000$ p.s.i.

Our diagram V gives the value of $W_1 : W$ for $S = 2$ and $t = 20,000$ p.s.i. (form. 44).

Let us note that for $S = 2$ formulas 27 and 43 allow us to write

$$\frac{\Delta_1}{\Delta} = 2 - \frac{t_1}{t}$$

Practical Solution of the Fundamental Problem

We call the fundamental problem, the one where the value given for A_r and A_c by the diagram I to III or the corresponding formulae is acceptable.

The values of F_p and F_q being given, we know m ; we accept $t = 20,000$ p.s.i., and $t_c = 140,000$ p.s.i., we require $S = 2$. We know that it is advisable to accept a value of t' as high as possible, say $t' = 20,000$ p.s.i.

The diagram IV gives t_1 (formula 45) and the formulas (20) to (23) give the solution.

Let us suppose, for example

$$F_p = 100 \text{ t.} \quad F_q = 200 \text{ t.}; \text{ hence } m = 2$$

Diagram IV gives $t_1 = 4,000$ p.s.i. and the formulas (20) to (23) give

$$A_r = 14.20 \text{ sq. in.} \quad A_c = 4.24 \text{ sq. in.}$$

$$P_1 = 229 \text{ t.} \quad P_1 = 46 \text{ t.}$$

Consequently (as $A = 33.6$ sq. in.)

$$\frac{W_1}{W} = 0.55 \quad \frac{C_1}{C} = 0.802 \quad \frac{\Delta_1}{\Delta} = 1.80$$

Practical Solution in case the A_r found is not Acceptable

In the case where A_r given by the fundamental theory is not acceptable, we have to choose a value of A_r but in this case we can no longer choose t' arbitrarily.

From formulas (28) to (30) is deduced after having replaced

t'_w by t'

$$t' = \frac{F_q}{A_r} - t_1 - \frac{F_q}{A_r} \frac{I}{I + \frac{A_r t_c}{F_p + F_q - t_1 A_s}} \quad (46)$$

Taking into account equation (43) written for $S = 2$, this becomes

$$(2t - t_1) \frac{m}{I + m} = \frac{F_q}{A_r} \frac{A_r t_c}{F_p + F_q - t_1 A_r + t_c A_r}$$

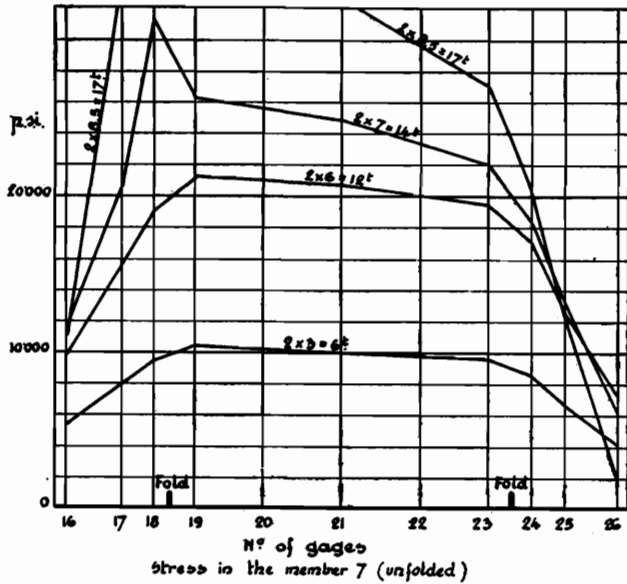


Fig. 8

This is an equation in t_1 which gives

$$t_1 = \frac{1}{2} \left[(2t + t_c + (I + m) \frac{F_p}{A_r}) - \sqrt{[2t + t_c + (I + m) \frac{F_p}{A_r}]^2 - 8t \left[(I + m) \frac{F_p}{A_r} + t_c \right] + 4(m + I) \frac{F_p}{A_r} t_0} \right] \quad (47)$$

Consequently, our problem is solved as follows :
Compute t_1 from 47 ; t' from 46, say

$$t' = \frac{2 m t - (I + 2 m) t_1}{I + m} \quad (48)$$

Apply then formulæ (28) to (31).

We have drawn the diagrams VI and VII, which translate the formulas (47) and (48).

Let us take the following example :

$F_p = 100$ t. ; $F_q = 25$ t. ; hence $m = 0.25$

Let us choose $A_r = 3.74$ sq. in.

Formula (47) gives $t_1 = -7,230$ p.s.i. ; and

formula (48) $t' = 16,700$ p.s.i.

Finally, through (28) to (31)

$A_e = 2.14$ sq. in.

$P_1 = 128.3$ t. $\Delta P_1 = 9.1$ t.

In this case ($A = 14$ sq. in.)

$$\frac{W_1}{W} = 0.422 \quad \frac{C_1}{C} = 0.730 \quad \frac{\Delta_1}{\Delta} = 2.36$$

Practical Calculation, when $\Delta_1 : \Delta$ is given together with $S = 2$

Formulas (32) and (43) combined (with $S = 2$) give

$$\frac{\Delta_1}{\Delta} = \frac{2t - t_1}{t} \quad (49)$$

Hence, if $\Delta_1 : \Delta$ is given, we know t'

$$t' = t \left(\frac{\Delta_1}{\Delta} \frac{2m + I}{m + I} - 2 \right) \quad (50)$$

Consequently, the problem is solved as follows :

Compute t' from (50) and then apply the formulæ (34) to (37) by giving to t'_w the value found by (50).

Let us take the following example :

$F_p = 100$ t. ; $F_q = 25$ t. Hence $m = 0.25$.

Let us take $\Delta_1 : \Delta = 2$.

Formulas (50) and (45) give

$$t' = 8,000 \text{ p.s.i. } \quad t_1 = 0.$$

From formulas (34) to (37) with $t'_w = 8,000$ p.s.i., we find

$A_r = 5.0$ sq. in.

$A_e = 1.95$ sq. in.

$P_1 = 118.0$ t.

$\Delta X_1 = 7.0$ t.

In this case

$$\frac{W_1}{W} = 0.50$$

$$\frac{C_1}{C} = 0.78$$

$$\frac{\Delta_1}{\Delta} = 2$$

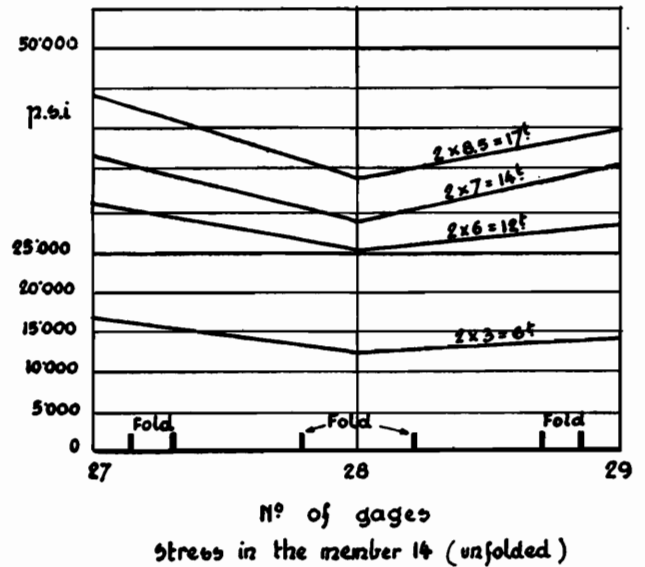


Fig. 9

Results of the Test of a Trussed Girder with a Prestressed Lower Member

Fig. 2 gives the theoretical shape of the girder ; Fig. 3 shows a photo of it ; Fig. 4 gives the cross-section of the different elements.

The lower member has been prestressed by means of a cable of 16 wires of 5 mm. (0.196 in.) having the following characteristics :

Ultimate	248,000 p.s.i.
Equivalent yield stress	220,000 p.s.i.
Modulus	27,800,000 p.s.i.
Elongation on 7 diam.	11.9%

The steel of the trussed girder itself is commercial mild steel with an ultimate strength between 53,000 and 60,000 p.s.i.

The beam has been prestressed after the deadweight of 2,210 lb. has been applied. The average stress established in the wires is 92,800 p.s.i., giving a total initial prestressing force of $P_1 = 92,800 \times 0.485 = 44,900 \text{ lb.} = 20.4 \text{ t.}$

The anchorage of the wires at the end of the beam is shown in Fig. 5.

Fig. 6 shows the loading system consisting of a steel girder placed on top of the prestressed girder at the points corresponding to the vertical elements 28 and the symmetrical one; this loading girder carries a jack on top of which we have placed a hydraulic load measuring device giving an accuracy of 1 per cent.

Strain measurements have been taken by means of electrical strain-gauges numbered 1 to 34 and placed as shown in Fig. 7.

We have also measured the deflection at midspan and the change in length of the lower member.

For the sake of simplicity we shall only record here the results given by the loading test and compare them with the theoretically computed values.

Let us however first mention the stresses as computed in the system under the separate and combined effect of the dead load and the prestressing; Table II gives these values. We have added to it a column giving the calculated stresses under the load of $2 \times 1 \text{ t.}$

TABLE II
(Stresses in p.s.i.)

Element		Stress under dead load	Stress due to prestressing	Combined stresses	$2 \times 1 \text{ t}$
Top member	1	-285	0	-285	-496
" "	7	-855	0	-855	-3,050
Bottom member	8	0	-12,000	-12,000	-327
" "	14	+1,700	-19,000	-17,300	+5,500
Diag.	15	+1,140	0	+1,140	+2,330
Diag.	21	+142	0	+142	0
Vertical	23	-1,280	0	-1,280	-2,630
Deflection		+0.094 in.	-0.625 in.		+0.281 in.
Shortening of lower member		-	+0.320 in.		-0.052 in.

One indication of this Table I may seem wrong at first sight, namely, the fact that under $2 \times 1 \text{ t.}$ the stress in the end element 8 of the lower member is in compression, whereas in an ordinary trussed girder, it would be zero. This is due to the fact that, as the girder deflects, the bottom member elongates and consequently the stress in the cable increases; this increase for $2 \times 1 \text{ t.}$ is 0.543 t.

With the results of Table I, we can now compute the working load of the girder and the load producing the yield stress in the lower member.

We will assume that $t_1 = 10,000 \text{ p.s.i.}$, and that the yield point of the mild steel is equal to $2 \times 20,000 = 40,000 \text{ p.s.i.}$

We must also mention that the loading device (jack, girder, etc.), weighs 0.4 t. (896 lb.).

The working load $2 \times P_w$, giving $t_1 = 10,000 \text{ p.s.i.}$, is given by (see in Table II stresses in lower member 14).

$$17,300 + 10,000 = P_w \times 5,500.$$

Hence

$$P_w = 5 \text{ t.}$$

It should be remembered that P is one of two point loads on the girder and not the total load.

The load $2 \times P_e$ producing the yield stress in element 14 can be computed as follows:

$$17,300 + 40,000 = P_e \times 5,500.$$

Hence

$$P_e = 10.40 \text{ t.}$$

This means that the calculated safety factor is

$$S = \frac{10.40}{5} = 2.08$$

Under the load $2 \times P_e = 2 \times 10.40 \text{ t.}$, the prestressing force has become

$$20.4 + 0.543 \times 10.40 = 26.1 \text{ t.}$$

giving a stress in the wires of 120,000 p.s.i.

The stress in the vertical element 23 is

$$-1,280 - 2,630 \times 10.40 = -28,700 \text{ p.s.i.}$$

The stress in the top member 7 is

$$-855 - 3,050 \times 10.40 = -32,555 \text{ p.s.i.}$$

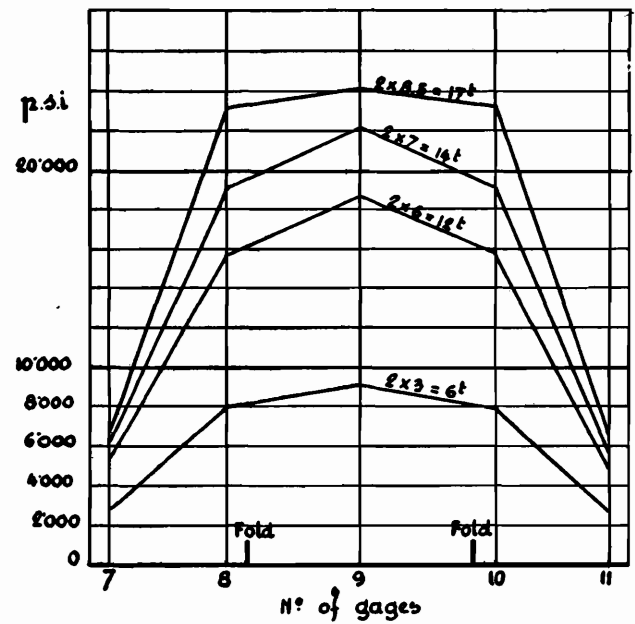
However, the stresses in these latter compressed elements must be divided by the buckling factor and become

for vertical element 23

$$-28,700 : 0.83 = -34,600 \text{ p.s.i.}$$

for top member 7

$$-32,555 : 0.88 = -36,800 \text{ p.s.i.}$$



Stress in the diag. 15 (unfolded)

Fig. 10

These figures show that under a load of $2 \times 10.40 \text{ t.}$ the bottom member, the first vertical element and the top member are all under stresses very near to the yield stress. If we learn now that we have taken no account in these calculations of the possibility of the top member buckling as a whole, it is understood that the girder will fail owing to this latter cause.

Table III gives a summary of the calculated stresses.

TABLE III

		Prestr. + dead w. $+2 \times 5 \text{ t.}$	Prestr. + dead w. $+2 \times 10.40 \text{ t.}$
Top member	7	-18,300	-36,800
Lower member	14	+10,000	+39,500
Vertical	23	-17,400	-34,600
Diagonal	15	+12,700	+25,200
ΔP_1 in %		13.3	27.8
Cable stress		105,000	117,000
Deflections in in.		0.87	2.40

NOTE. Stresses are in p.s.i.; compressive stresses are increased by buckling factor.

We are now going to make a comparison between the measured strains and deflections and the calculated ones. In transforming the strains measured into stresses, we have used for mild steel a modulus of 29,000,000 p.s.i.

In Table IV we compare the values obtained during the variation of P from zero to 6 t.

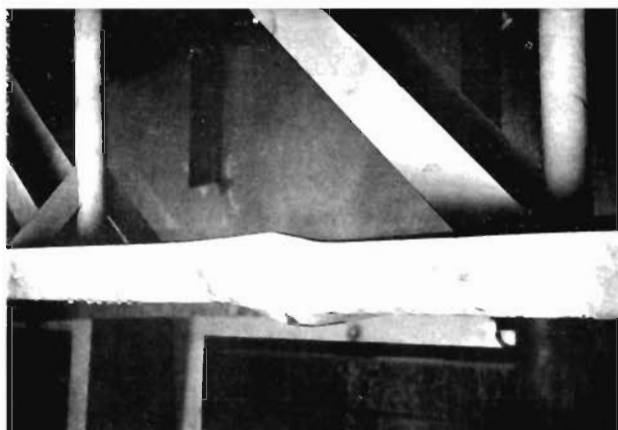


Fig. 11.—Central part of beam after failure by buckling

TABLE IV
(P from 0 to 6 t.)

	Strain-gauge	Stresses		
		Calculated	Measured	
Top member No. 1	1	- 3,000	- 2,560	average - 2,750
	2	- 3,000	- 3,130	
	3	- 3,000	- 2,560	
Lower member No. 8	4	- 2,000	- 3,850	average - 2,270
	5	- 2,000	- 2,000	
	6	- 2,000	- 1,000	
Diagonal No. 15	7	+ 13,900	+ 5,400	average + 12,800
	8	+ 13,900	+ 15,600	
	9	+ 13,900	+ 18,400	
	10	+ 13,900	+ 15,900	
	11	+ 13,900	+ 24,840	
Vertical No. 23	12	- 15,800	- 12,800	average - 11,900
	13	- 15,800	- 12,400	
	14	- 15,800	- 11,100	
	15	- 15,800	- 11,100	
	16	- 18,100	- 9,800	
	17	- 18,100	- 15,300	
	18	- 18,100	- 18,900	
Top member No. 7	19	- 18,100	- 21,100	average - 17,500
	20	- 18,100	- 20,800	
	21	- 18,100	- 20,700	
	22	- 18,100	- 20,000	
	23	- 18,100	- 19,300	
	24	- 18,100	- 17,000	
	25	- 18,100	- 12,600	
	26	- 18,100	- 7,250	
	Lower member No. 14	27	+ 32,800	
28		+ 32,800	+ 25,300	
29		+ 32,800	+ 29,000	
Diagonal No. 21	30	0	- 25,500	average 0
	31	0	+ 850	
	32	0	+ 22,700	
	33	0	+ 12,800	
	34	0	- 2,000	

In Table V the comparison is made for the variation of P from zero to 8.5 t.

These results suggest the following remarks :

I.—For P varying from zero to 6 t.

a.—As a whole the measured stresses are fairly close to the calculated ones.

b.—The greatest difference is for the vertical element No. 23, which carries less than the calculation shows. This is probably due to the stiffness of the joints which, in the calculation, are supposed to be frictionless pin joints.

c.—The lower member No. 14 carries about 13 per cent. less than calculated; this is probably due to the fact that the construction joint at the middle of the span is a bolted joint.

II.—For P varying from zero to 8.5 t.

a - b - c as above.

d.—In the top member No. 7 some strain-gauges have given strains above the elastic limit; notwithstanding this we have multiplied them by the value of E corresponding to the elastic state; but this gives fictitious stresses, which are far above the real ones; this is the reason why the average stress in this element is 15 per cent. higher than the calculated one.

III.—In general it may be noted that the stresses are far from uniform in the cross-section of the different elements even in the middle of these elements, where the bending moment must be very small.

In view of illustrating this, we have prepared Figs. 8, 9 and 10 giving for different loads the stress distribution in the cross-section of elements 7, 14 and 15.

TABLE V
(P from 0 to 8.5 t.)

	Strain-gauge	Stresses		
		Calculated	Measured	
Top member No. 1	1	- 3,980	—	
	2	- 3,980	—	
	3	- 3,980	—	
Lower member No. 8	4	- 2,850	—	
	5	- 2,850	—	
	6	- 2,850	—	
Diagonal No. 15	7	+ 19,800	+ 6,800	Average + 17,200
	8	+ 19,800	+ 23,000	
	9	+ 19,800	+ 27,000	
	10	+ 19,800	+ 23,400	
	11	+ 19,800	+ 6,520	
Vertical No. 23	12	- 22,400	- 16,900	Average - 16,200
	13	- 22,400	—	
	14	- 22,400	- 15,600	
	15	- 22,400	—	
Top member No. 7	16	+ 25,600	- 10,800	Average - 29,700
	17	+ 25,600	- 34,600	
	18	+ 25,600	- 67,000	
	19	+ 25,600	- 61,800	
	20	+ 25,600	—	
	21	+ 25,600	- 32,500	
	22	+ 25,600	—	
	23	+ 25,600	- 26,600	
	24	+ 25,600	- 20,500	
	25	+ 25,600	- 12,100	
26	+ 25,600	- 2,000		
Lower member No. 14	27	+ 46,600	+ 43,000	Average + 39,800
	28	+ 46,600	+ 34,300	
	29	+ 46,600	+ 40,500	
Diagonal No. 21	30	0	—	
	31	0	—	
	32	0	+ 1,700	
	33	0	—	
	34	0	—	

Fig. 8 shows clearly that the yield point is reached at the point occupied by the strain-gauge No. 18 for the load of 2×7 t.; this was the beginning of the failure of the beam by the buckling of the top member as a whole.

The maximum load reached was 2×8.5 t. This failure has nothing to do with the prestressed lower member in which the yield stress would have been reached according to calculations under a load of 2×10.40 t.

It should be noted that for the load of 2×7 t. the calculated stress in the element No. 7 is (see Table II).

$$-853 - 7.2 \times 3,050 = -22,855 \text{ p.s.i.}$$

or, taking the buckling into account over the length of one panel

$$-22,855 : 0.88 = -26,000 \text{ p.s.i.}$$

which is far below the yield stress. But the failure occurred not by this stress being too high, but by the buckling of the top member over its total length.

We intend to repeat the test on a similar beam, in which the general buckling will be avoided by side guiding.

Fig. 11 gives a photo of the central part of the beam after failure by buckling.

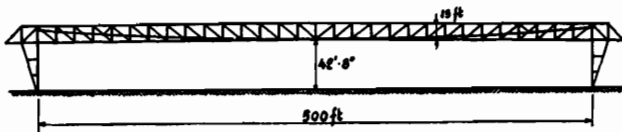
Let us finally mention the following results in inches :

	2×6 t.		2×8.5 t.	
	Calc.	Meas.	Calc.	Meas.
Deflection	1.69	2.00	2.38	3.58
Elongation of cable	0.31	0.32	0.44	0.51

The differences between the measured and calculated values are due to the bolted joint as already explained, and for the load of 2×8.5 t. at the same time to the fact that the yield stress is reached in the top member.

Practical Example

Fig. 12 gives the general cross-section of an aeroplane hangar made in prestressed structural steel. It has a span of 500 ft. and the beams—8 ft. apart—have only a depth of 15 ft. It is the most economical structure in the present state of engineering.



Cross section of an aeroplane hangar made in prestressed structural steel.

Fig. 12

Other Ideas on Prestressed Structural Steel

The idea of prestressing steel has been considered during the last few months. The most interesting paper on it was published in Germany by Dr. Ing. F. Dischinger, Professor at the Technical University of Berlin, at the end of last year (*Der Bauingenieur*, Heft 11 and 12; 1949). He discusses the method proposed by us and concludes that the saving it can give is moderate. We do not agree with his arguments, which are twofold :

a.—By replacing part of the lower flange of a mild steel girder by high tensile steel, the girder is weakened and the stresses in it due to the live load increase. This obviously is an incomplete view.

b.—The tensile force developed by the live load in the lower flange, is very different from the compressive force developed in the top flange. This is true, but what does it matter? What does it mean in relation to the cost of the girder? It may well be that Dr. Dischinger is correct in stating that his method of having the concrete road slab working together with the main steel girders of a bridge, gives a larger saving than our

proposal to prestress the lower member of the girders, but this is another problem. It shows only that the greatest saving would be reached by applying both methods simultaneously.

What Dr. Ing. Dischinger proposes is to consider steel girders with a concrete slab solidly attached to their top flange, the whole of this compound structure being then prestressed by any of several means he suggests.

In his idea the concrete incorporated is nothing else but the concrete slab used as support of the road or the railway tracks of a steel bridge. This concrete slab exists in any case in a steel bridge, but up to the present it has never been taken into account as working together with the main steel girders.

He first explains that the concrete slab cannot be relied upon as working together with the steel girder, unless it be sure that it is not cracked by shrinkage or by the negative moments existing above the internal supports, if the beam is a continuous one over more than two supports. Hence his first requirement is that the slab should be prestressed to such an extent that it never cracks due to loads or to shrinkage. This is the limited scope of the prestressing proposed by him.

The slab can be prestressed in several ways; for example :

a.—by casting it on top of the steel girder, whilst this girder is still supported by formwork underneath. If this formwork is removed after the concrete has hardened, the slab will be prestressed; however, this prestress is maximum at midspan and zero at the ends, if the girder is simply supported, consequently the result is not perfect.

b.—By casting the slab on top of the beam, and then—in case of a continuous beam—by lowering slightly all the supports except the extreme ones. This has the disadvantage of prestressing the slab as well above the supports, as at the middle of the spans; consequently a smaller compressive stress remains available at midspan for the live-load.

The system which Dr. Ing. Dischinger recommends is to cast the slab above the steel girder, but on a steel-plate, which is kept at a distance from the top flange of the steel girder by round transverse rods or rollers, allowing a free longitudinal movement. When the slab has hardened, he prestresses it by cables attached to its ends; when this prestressing is done, he fixes with rivets the steel plate on which the slab has been cast to the top flange of the girder. He can then remove the prestressing cables or not, and place cement mortar in the gap kept open by the round rods.

In large continuous beams, he does not only use a concrete slab on top of the girder, but provides one underneath the bottom flange near the supports, so as to increase the moment of inertia of the cross-section.

Here we must leave this description of principles. The general conclusion is that the idea of prestressing steel structures is gaining ground and is bound to develop.

The specialists in structural steel have the great advantage of having at their disposal proper means of prestressing. The girder tested in our laboratory has been prestressed with the same sandwich cable, which we are using for prestressed concrete and with the same jacks. This fortunate fact will allow prestressed steel to make much quicker progress than has been possible in prestressed concrete.